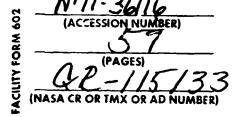
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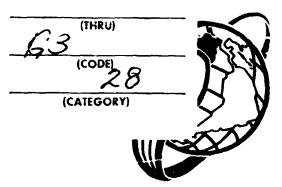
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EXTENDABLE NOZZLES FOR SPACE ENGINES

Report 10484-FRA

30 July 1971







AEROJET LIQUID ROCKET COMPANY

SACRAMENTO, CALIFORNIA

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EXTENDABLE NOZZLES FOR SPACE ENGINES

Prepared Under
Contract NAS 9-10484

for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION
Manned Spacecraft Center
Houston, Texas

AEROJET LIQUID ROCKET COMPANY
Sacramento, California

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FOREWORD

This report, whose purpose is to present the findings of analysis of the extendable nozzles for space engines test program, is submitted in partial compliance with the requirements of Contract NAS 9-10484.

The contents of this report completes the technical effort and satisfies the program objectives as defined by the contract and is intended to verify and/or upgrade Volume I (Program Studies) and Volume II (Design Guide) of the Final Report (10484-FR).

All work under the subject contract was performed for the National Aeronautics and Space Administration's Manned Spacecraft Center, Houston, Texas, by the Aerojet Liquid Rocket Company. The program was under the direction of Dr. N. E. Van Huff, Program Manager; R. C. Schindler, Project Manager; and E. Schmauderer, Project Engineer. The NASA Project Engineer for the program was Mr. G. Hubbard.

I. INTRODUCTION

The objective of the Extendable Nozzles for Space Engines Program was to prepare a comprehensive design guide for extendable/retractable nozzle extensions for space engine applications.

The program was conducted in two overlapping phases, extending from 12 January 1970 to 30 July 1971. The first phase was devoted to engineering analysis and the design of an experimental thrust chamber incorporating an extendable/retractable nozzle extension. The second phase consisted of fabrication of the Phase I thrust chamber design, continued engineering analysis, and delivery of the thrust chamber to the NASA White Sands Test Facility where it was to be tested. The data derived from the experimental testing was to be used in conjunction with or to upgrade the results of analysis. Delays in testing resulted in preparation of the final report and design guide without benefit of the experimental data derived from testing.

This document presents the findings of analysis of the experimental test data, upgrading the Final Report (10484-FR).

II. SUMMARY

This section serves to summarize the conclusions and recommendations derived from analysis of the experimental extendable/retractable nozzle test data and to evaluate the impact of these results on Volume I (Program Studies) and Volume II (Design Guide) of the final report.

A. SUMMARY OF ANALYTICAL CONCLUSIONS

The conclusions resulting from the test data analysis are as follows:

- Heat flux in the area of gas reattachment downstream of a rearward facing step increased significantly as height of the step was increased. An increase in heat flux of up to 2.5 times that for a smooth wall was measured (Figure 1).
- Nozzle performance, I sp(del), significantly reduced as the height of a rearward-facing step was increased (Figure 2).
- Supplemental supersonic film cooling (GN₂) of the hot gas reattachment zone on the nozzle wall downstream of a rearward facing step was highly effective. A minimum amount of film coolant was required to reduce the uncooled heat loads (Figures 3, 4 and 5).
- Of the two nozzle sealing concepts tested, the static face seal (as recommended in the final report) is the most desireable for incorporation into an extendable/retractable nozzle design.
- Nozzle de loyment and retraction, with the engine firing, was successfully demonstrated under simulated altitude conditions.

II, Summary and Comparison of Conclusions (cont.)

B. IMPACT OF ANALYSIS ON FINAL REPORT

1. Heat Transfer

At the point of gas reattachment in the step region of the extendable nozzle, the design guide recommended that the maximum heat flux be calculated using Newton's law of cooling and an amplified heat transfer coefficient. The heat flux would therefore be calculated as

$$q = h(T_{wa} - T_{w})$$

where

q = heat flux

h = step region film coefficient

T = adiabatic wall temperature

T = nozzle wall temperature

An amplification factor of 3.0 was recommended, so that the step region film coefficient equation became:

$$h = 3.0 hg$$

where

3.0 = recommended amplification factor

*hg = smooth wall film coefficient

The step region coefficient, h, was not determined experimentally during the extendable nozzle test program since the calorimeters needed to supply data used in calculating this parameter did not operate properly. However, using the 3.0 amplification ratio, it can be stated

$$\frac{q}{q_0} = 3.0$$

^{*}Normally calculated for rocket nozzles using Bartz equation or an equivalent correlation.

II, B, Impact of Analysis on Final Report (cont.)

where q = step region heat flux $q_0 = \text{smooth wall heat flux}$

Accordingly, a heat flux ratio of film cooled to smooth wall flux might also be used for design. The experimentally determined nozzle heat flux data showed that the maximum step region heat flux dependent on step size, and the amplification factor varied between 1.0 and 2.5.

$$\frac{q}{q}$$
 = 1.0 to 2.5*

Therefore, it is concluded from these data that an amplification factor of 3.0 applied to either the smooth wall film coefficient or the smooth wall heat flux will give a conservative estimate of step region heat transfer.

With film cooling, Newton's law of cooling was also recommended with an amplification factor of 3.0 applied to the film coefficient as before. The adiabatic wall temperature with film cooling is caluclated using Goldstein's correlation:

$$\eta = \frac{T_{\text{wa}} - T_{\text{c}}}{T_{\text{r}} - T_{\text{c}}} = 1.0, \ \xi \le 15.5$$

$$\eta = \left(\frac{15.5}{\xi}\right)^{2.5}, \ 15.5 < \xi < 39.0$$

^{*}Based on experimental smooth wail data.

II, B, Impact of Analysis on Final Report (cont.)

where

$$\xi = (\frac{x}{h}) \cdot (\frac{T_c}{T_c}) \cdot (\frac{1}{M})^{0.27}$$

x = film cooled length

h = coolant slot height

 T_{O} = mainstream gas stagnation temperature

 $T_c = coolant supply temperature$

M = ratio of coolant to mainstream mass velocity

 T_{wa} = adiabatic wall temperature

T = free stream recovery temperature

Series III, Test-003 data are compared to this prediction in the step region and downstream of the step in Figure 7. The predicted heat flux was calculated with the recommended amplification factors of 3.0 and 1.0, and these curves are shown in the figure. Comparison of the data to these predictions shows that the 3.0 amplification factor is overly conservative in the step region where a 1.0 factor appears to be adequate. Also, downstream of the step, the prediction with a 1.0 amplification factor is seen to provide a conservative estimate of the film-cooled wall heat flux.

2. Performance

The Design Guide recommendation, that a deployable nozzle should be designed for a minimum of discontinuity at the interface between the fixed portion of the nozzle and the movable extension to minimize the loss in performance $(I_{sp(del)})$ resulting from discontinuities in the nozzle wall, was verified during this study. The experimental data, Figure 2, shows that the loss in nozzle performance versus step height (wall discontinuity) increases with step height.

At the time the Final Report was prepared, neither analytical nor experimental data was available for predicting the magnitude of nozzle losses resulting from discontinuities; these losses were assumed to be small for properly designed joints, i.e., joints with minimum discontinuity.

II, B, Impact of Analysis on Final Report (cont.)

Consequently, the performance data presented in the rinal report does not include losses attributable to nozzle wall discontinuities.

The delivered performance $(I_{sp(del)})$ for each nozzle configuration presented in the Design Guide was estimated by calculating the one-dimensional kinetic vacuum specific impulse and subtracting the various real engine losses using the following equation:

$$I_{sp(de_{\perp})} = I_{sp(odk)} - I_{sp(er/mrd)} - \Delta I_{sp(bl)} - \Delta I_{sp(div)} - \Delta I_{sp(c)}$$

where

wp(odk) = one-dimensional kinetic vacuum specific impulse
based on injector mixture ratio

 $^{\Delta I}$ sp(er/mrd) = specific impulse loss due to energy release and mixture ratio maldistribution

 $\Delta I_{sp(bl)}$ = specific impulse loss due to boundary layer effects

 $\Delta I_{sp(div)}$ = specific impulse loss due to flow divergence at the nozzle exit

 $\Delta I_{sp(c)}$ = specific impulse loss due to nozzle extension cooling

The use of this equation is restrictive and should be expanded to include an estimate of the nozzle performance losses associated with nozzle wall discontinuities. Unfortunately, the loss in $I_{\rm sp(del)}$ as presented in Figure 2, is indicative of the uncooled nozzle losses experienced with the experimental test engine only. Close inspection of the uncooled nozzle performance data does indicate that even small nozzle wall discontinuities produce performance losses and that the engine designer should not assume these losses equal to zero, no matter how small the wall discontinuity. Care should be taken by the designer when attempting to predict losses associated with small nozzle wall discontinuities, for it is conceivable that, as the attachment area ratio becomes smaller, the performance loss increases for a given wall discontinuity. That is to say, that the $C_{\rm f}$ for a given nozzle configuration is a larger function of the smaller area ratios and not so much the larger

II, B, Impact of Analysis on Final Report (cont.)

area ratios, and that a wall discontinuity at a small area ratio would produce a larger performance loss than the same discontinuity at a larger area ratio. Therefore, a nozzle performance loss for discontinuities up to 0.25 in. could reach 2.0 lb-sec/lb or more. Another factor to consider is that the performance loss presented in Figure 2 is associated with the configuration of the discontinuity tested and, for a discontinuity of a different configuration — even at the same area ratio — the magnitude of performance loss could be greater or less.

3. Nozzle Sealing and Supplemental Cooling

Evaluation of the data derived from the test program did not yield additional design data to upgrade that presented in the final report.

The suggested design techniques, as presented in the final report, remain valid.

4. Nozzle Deployment and Retraction

No data adversely affecting the design recommendations established and presented in the final report were developed during evaluation of the nozzle deployment and retraction system. The recommended design approach and considerations, as presented in the final report, remain valid.

III. TEST PROGRAM

A. OBJECTIVE

The objectives of the experimental extendable nozzle test program were (1) to evaluate the effect of contour mismatch at the nozzle attachment interface on heat transfer and performance, (2) to evaluate the feasibility of film cooling at the nozzle attachment interface, (3) to evaluate sealing of the interface between the fixed and movable portions of the nozzle, and (4) to demonstrate the operation of an extendable - retractable rozzle extension with the engine firing under simulated altitude conditions.

B. DESCRIPTION OF TEST HARDWARE

To satisfy the test program objectives, the test engine was configured to permit evaluation of: (1) a charge in step heights; (2) the addition of auxiliary film cooling of the nozzle extension downstream of the divergent step, as well as various sealing concepts; and (3) nozzle deployment and retraction. The test engine shown in Figure 6 was designed around the NASA-supplied LEM workhorse injector and chamber. This limited the quantity of new components to those areas which were related to the extendable nozzle. The new designs consisted of a chamber - nozzle adapter, a nozzle film coolant injector, nozzle deployment and retraction system, and a nozzle extension.

1. Nozzle Extension

The nozzle extension (heat-sink design) was made of 347 stain-less steel with a constant wall thickness of 0.200 in. It had a cylindrical section several inches long so as to put the attachment flange in a cooler area and provide interfaces to accept the several different sealing concepts to be evaluated. Nine calorimeters, 24 thermocouples, and 14 pressure taps

III, B, Description of Test Hardware (cont.)

were used to monitor thermal and pressure conditions throughout the extension. A Rao contour was selected for the entire nozzle assembly and the rigid, heat-sink extension duplicates this contour from an area ratio of approximately 22:1 out to 37.1.

Instrumentation was concentrated in the predicted hot gas reattachment locations on the nozzle wall for the different fixed-nozzle step conditions to be evaluated (Figures 8 and 9).

2. Chamber-Nozzle Adapter

The chamber - nozzle adapter was designed to physically bridge the area from the LEM chamber to the movable nozzle extension. The adapter served as the backbone of the design, supporting all other components and serving as interface between the entire assembly and a thrust takeout or measuring system provided by the test facility. Further, it provided the capability of delivering a secondary cooling flow to the reattachment area and the inner wall of the nozzle extension for evaluation of this cooling technique. It also supported the nozzle guide and actuation systems required to translate the nozzle, and served as a support for the seals necessary to seal the moving and fixed portions. Finally, it provided a variable geometry at the end of the fixed portion of the tip in order to determine the effect of different fixed-to-movable transition step heights or configurations (Figures 6 and 10).

3. Film Coolant Injector

The supersonic injector used for injecting nitrogen into the reattachment area was an integral part of the nozzle-chamber adapter. The injector not only provided evenly distributed coolant by 180 two-dimensional nozzles but incorporated removable tips (as shown in Figure 11) to provide

III, B, Description of Test Hardware (cont.)

discontinuities in the nozzle wall of zero in., 0.25 in., 0.50 in., and 0.75 in. for testing in both cooled and uncooled states. The coolant injector provided coolant mass flow control from zero lb/sec to 0.5 lb/sec to the step region. The coolant injection velocities ranged from 1291 ft/sec at 0.1 lb/sec flow to 1485 ft/sec at 0.5 lb/sec.

4. No :zle Deployment and Retraction System

Three separate items made up the nozzle deployment and retraction system. They were the actuation system, the guide and support system, and the nozzle slide section (Figure 6).

a. Nozzle Actuation System

The nozzle actuation system was composed of three singlerod hydraulic actuators. The actuators connected directly the movable nozzle
to the engine frame and distributed loads equally about the three rails used
in the guide system. Actuation speed was controlled by matched orifices in
the hydraulic supply and discharge lines.

b. Guide and Support System

The guide and support system functioned as the name implies -- support the moving hardware (in this case, the slide section and exit cone attached to it) at all times under all loads, whether extended or retracted, and guide it so as to a hieve proper radial clearances and alignment when deployed.

The guide/support system selected for use was the Thomson Industries' ball bushing system. The rails or guides consisted of case hardened centerless ground steel rods which were mounted to the three

III, B, Description of Test Hardware (cont.)

support gussets of the chamber-nozzle adapter by bolts and spacers, allowing shims to be added for final adjustment of nozzle alignment. The ball bushing was open, permitting the rail stands to pass through the longitudinal slot in the bushing.

The bushing is a housing which supports rows of iongitudinally recirculating ball bearings which roll between the shaft and the housing, contacting the shaft around 75 to 80% of its circumferential surface. The housing (bushing) was, in turn, held and clamped in the hardware to be moved (slide section and extension).

c. Slide Section

The nozzle slide section (as shown in Figure 6) served four distinct functions:

- (1) It held the bushings which ride the guide rails and supported them.
- (2) It served as a fixture to which the nozzle extension was mounted and sealed.
- (3) It served as the movable sealing surface for the fixed-to-moving nozzle seals.
- (4) It was the point of attachment for the actuation system.

This ring was designed to transmit all transverse loads on the extension to the guides (six roller bashings) and all longitudinal loads to the actuation system.

III. B. Description of Test Hardware (cont.)

5. <u>Fixed-to-Movable Nozzle Seals</u>

Two different sealing concepts were incorporated into the test engine design for evaluation during the test program. They were the concept of face sealing (static) and piston-type (dynamic and/or static) or diametral sealing.

The face seal (as shown in Figure 6) was installed at the interface between the nozzle slide section and the chamber - nozzle adapter. The small flange on the ID of the slide section served as the sealing surface for the conventional O-ring mounted in the forward-looking face of the nozzle adapter.

•

The piston-type or diametral seal was also located at the interface between the nozzle slide section and the chamber-nozzle adapter. The adapter again provided the O-ring groove on its OD of the fixed part and the ID of the slide section served as the seals mating sealing surface. The elastomeric seal satisfying the requirements for a diametral seal, that the seal requires a minimum force to compress, will seal large radial clearances between mating pars, and will not tend to roll out during relative part movement was the O-ring elastomeric seal (Figure 12). The seal was glued into the O-ring groove on the fixed part to prevent roll-out and was hollow to minimize compression forces required for good sealing.

Both seals, the face and diametral, were made of silicone rubber for use in a high operating temperature environment.

C. EXPERIMENTAL TESTING

The test program, conducted at the NASA/WSTF in Las Cruces, New Mexico, was initiated on 19 February 1971 and concluded on 14 April 1971. During this period, a total of 25 engine tests were conducted at simulated altitude conditions using N_2O_4 and AeroZINE 50 propellants at a nominal mixture

III, C, Experimental Testing

ratio of 1.6 and chamber pressure of 100 psia. Photographs of the test engine mounted in the testing facility are shown in Figures 13, 14, and 15. Seven of these tests were classified as system balance and data point verification tests, while the remaining 18 tests satisfied the program objectives. The test data for the 18 tests are summarized in Table I. A summary of the 25 engine fire test is enclosed as Table II. The detailed test plan was included as Appendix B of the final report for this program, Extendable Nozzles for Space Engines, Volume I, Report No. 10484-FR.

The test program was divided into five discrete test series, each satisfying specific predefined test objectives. The test series and objectives are also shown in Table I.

Test Series No. I established the base performance of the test engine with a continuous divergent uncooled nozzle wall. This series consisted of two tests: a base data point and a verification data point.

Test Series II, III and IV were designed to evaluate nozzle wall discontinuities of 0.25 in., 0.50 in., and 0.75 in. at the interface between the extendable and fixed nozzle (shown in Figure 10) and consisted of four tests each. Three of the four tests evaluated the specific wall discontinuity, step height, at coolant flow rates of 0.1 lb/sec, 0.3 lb/sec and 0.5 lb/sec. The fourth test provided an uncooled base performance data point at that step height.

Test Series No. V investigated and demonstrated operation of the test engine with a partially deployed nozzle and deployment and retraction of the nozzle extension with the engine firing. Test Numbers V-001, V-002 and V-003 were the partially retracted tests, and Test No. V-004A was the deployment and retraction test.

III, Test Program (cont.)

D. TEST DATA ANALYSIS

1. Heat Transfer

The experimental test engine employed thermocouples, calorimeters, and gas probes (Figure 9) to obtain thermal data during the test program. The thermal instrumentation was designed to provide data which could be used to calculate the nozzle wall heat flux and the hot gas and GN₂ coolant film temperatures. Twenty-four thermocouples, nine calorimeters, and four gas probes were used to obtain the nozzle test data. The thermal instrumentation locations are summarized in Table III.

The calorimeters were designed to provide equilibrium nozzle wall temperature data that could be readily converted to nozzle wall heat flux. However, the data derived from testing indicated that the calorimeter did not operate properly. The instrument operated in a fashion similar to that of a thermocouple. It was concluded that the insulation between the calorimeter element and the main body was not adequate to prevent the transfer of heat to the body, resulting in the calorimeter responding in a fashion similar to a thermocouple. Therefore, these data were considered invalid and were not included in the thermal analysis. The gas probe temperature data were much lower than expected and were also considered to be in error and omitted from the analysis.

The test nozzle, as stated in Section III,B, was designed to provide nozzle wall discontinuity or step height at the nozzle extension attachment point and thermal test data for discontinuities of 0.25 in., 0.50 in. and 0.75 in. GN₂ film coolant was employed as an auxiliary coolant, and thermal data were obtained with 0.0, 0.1, 0.3 and 0.5 lb/sec film coolant flow. The test data for thirteen tests (indicated by asterisk in Table I) were analyzed.

III, D, Test Data Analysis (cont.)

The thermocouple temperature data were plotted for each test and smooth curves drawn to show the trend of temperature with axial distance. A typical plot for all series III tests (the 0.5-in. step height test case) is shown on Figure 16.

Selected data points were then taken from the temperature curves and utilized in calculating the nozzle wall heat flux. The method of heat flux calculation, which was developed at ALRC and uses the SINDA program, forces the wall surface temperature to follow the thermocouple transient data while calculating the heat flow through an arbitrarily large gas-side conductance. The calculated nozzle wall heat flux is plotted as a function of axial distance on Figures 3, 4 and 5, in which the step region heat flux is compared to both the smooth wall data and the Bartz predicted heat flux. Inspection of the plotted data, temperature and heat flux vs axial distance indicates the following:

- The smooth contour, zero step test (Test No. I-002) showed that the wall hear flux continually decreased with axial distance as was expected.
- The introduction of a step discontinuity in the contour produced a low heat flux region near the step base and a high-heat flux region near the shoulder and showed that this peak heat flux is highly dependent on the step height. The maximum calculated experimental heat flux (0.5 Stu/in. 2-sec) was experienced during test with the /5-in. step configuration. The experimental flux of 0.5 Stu/in. 2-sec is approximately 2.5 times the experimental smooth wall flux and is twice the heat flux predicted using the Bartz equation.
- The smooth wall data shown on Figures 3, 4 and 5 are 20 to 30% lower than the Bartz prediction. This

III, D, Test Data Analysis (cont.)

deviation of experimental to predicted heat flux was found to be normal and further verified the validity of the test data. Radiation-cooled nozzle temperature data from Apollo and Transtage testing indicate a heat flux approximately 30% lower than the Bartz predicted flux in the nozzle region (Figure 17).

• Relatively small film cooling flows were effective in cooling the step region. The 0.75-in. step data plotted on Figure 5 indicates that 0.1 lb/sec GN, was sufficient to alleviate the high local heating rates encountered without film cooling. The ${\rm GN}_2$ coolant flow of 0.1 lb/sec represents 1% of the nozzle mainstream gas flow. This is considered a small amount of coolant, particularly since GN, is considered a poor coolant by comparison to propellants. Figures 3 through 5 also show that the nozzle wall heating rates downstream of a step region are reduced as film coolant flows are increased. It should be noted that the peak heat flux in the step region is due to impingement of the hot gas which expands from the lip of the step and produces a very localized heat flux spike and that impingement of the hot gas on the film-cooled boundary layer did not completely destroy the effectiveness of the coolant but that the coolant continued to cool the nozzle wall downstream of the hot gas impingement point.

2. Performance

In a retractable - extendable nozzle there will be a small discontinuity between the movable extension and the fixed nozzle. The objective of the performance analysis was to identify the performance loss associated

III, D, Test Data Analysis (cont.)

with the nozzle discontinuities of 0.25 in., 0.50 in., and 0.75 in., as designed into the test hardware.

The nozzle performance (I_{sp}) was calculated and is presented in Table I. A plot of specific impulse vs. step height for both the uncooled and rilm cooled tests is presented as Figure 2. This data plot shows that a significant loss in performance was experienced. Also, that as the step height was increased, so did the loss in performance (I_{sp}) (up to 3 sec of impulse for a step height of 0.75 in. uncooled).

3. <u>Nozzle Sealing</u>

extension was demonstrated on all tests. The concept of a static face seal employing a conventional 0-ring satisfactorily sealed the fixed and movable nozzle interface throughout Test Series I through IV and Test V-004A. The same face seal as installed to initiate the test program was still in reuseable condition following 25 tests at the completion of the program. The static diametral sealing concept was satisfactorily demonstrated on Tests V-001, V-002, and V-003. No leakage was observed during any of the tests. Post fire visual inspection of the seals and mating surfaces showed no damage resulting from heat exposure. The face seal 0-ring was exposed to a hot gas environment for an accumulated testing duration of approximately 548 seconds. The static diametral seal exposure time was an accumulated 90 seconds.

The piston or diametral sealing technique for dynamic sealing of the interface between the fixed and movable sections of the nozzle extension proved to be unsatisfactory; however, this technique did provide good sealing when used as a static seal.

III, D, Test Data Analysis (cont.)

Buildup of the test engine included installation of the diametrial piston seal. Following a minimum number of nozzle actuations, during which the actuation system appeared to be marginal with respect to deployment and retraction resulting from the increased seal loads required for sealing, the seal was destroyed. Evaluation of the seal following cyclic operation of the nozzle extension indicated the following:

- The technique of glueing the seal into its gland proved satisfactory. The seal remained in its groove and was not rolled out during cyclic nozzle operation.
- A circumferential section on the OD of the seal, seal contact surface with the nozzle slide section, was torn out. This torn section was approximately 3 inches in length. It was concluded that even though the seal was hollow to minimize the compression forces required for good sealing, that the compression forces under dynamic conditions were great enough to cause seal failure. Not only did the seal fail, but the compression forces required for good sealing were sufficiently high enough to restrict cyclic operation of the nozzle extension (nozzle extension and retraction).

The conclusions drawn from a comparative view point of the test results of the two sealing concepts, remain as originally outlined in the final report.

4. Nozzle Deployment and Retraction

Demonstration of the nozzle deployment and retraction system was successfully accomplished by test during Test Series V.

III, D, Test Data Analysis (cont.)

Test No. V-004, originally slated to demonstrate the nozzle deployment and retraction system, was manually terminated after 16 sec of steady-state chamber pressure operation. The nozzle failed to return to its fully extended position during its re-extended cycle. The test sequence of events was as follows:

- Engine start
- At engine start + 5 sec, the nozzle translation system was signalled to retract the nozzle extension.
- The nozzle deployment and retraction system retracted the nozzle extension.
- At engine start + 10 sec, the nozzle translation system was signalled to extend the nozzle extension.
- The nozzle deployment and retraction system extended the nozzle but at a decreasing rate of travel until the ozzle, as viewed on the TV monitor, appeared to have stopped moving halfway through its re-extend cycle.
- The test was manually terminated at $FS_1 + 17$ sec

It was concluded following a detailed review of the test records and inspection of the test hardware that (1) the test hardware was in refireable condition (without damage); (2) that the hydraulic actuation pressure be increased to the actuators for the repeat test, if the test was to be repeated; (3) that partial demonstration of the nozzle deployment system did not totally demonstrate the system's capability; and (4) that the test should be repeated if possible.

The requirement of increased hydraulic actuation pressure to solve the deployment problem of the nozzle extension was proposed as a result of the identification of the following potential problem areas:

(1) That minor misalignment, resulting from nozzle pressure lcads, of the nozzle slide section to the guide rails could result in increased friction and resistance to nozzle travel, which could only be encountered with the engine firing.

III, D, Test Data Analysis (cont.)

- (2) That the bundle of instrumentation cabling attached to the movable nozzle (Figure 15) added sufficient additional resistance to prevent nozzle travel.
- (3) That the nozzle wall pressure deviations, as observed on Tests V-001 and V-002, from the predicted wall pressure profile contributed to prevent complete deployment of the nozzle extension (Figures 20, 21 and 22).
- (4) That all the above contributed in part to prevent fully deploying the nozzle extension.

Functional testing of the hydraulic actuation system revealed that several solenoid-piloted valves would not operate after being electrically powered for over five minutes, probably due to heating. The test procedure called for these valves to be electrically powered for approximately five minutes prior to test. The operational test procedure was modified to prevent unnecessary power application, and the hydraulic system functionally verified to be capable of reliable nozzle extension deployment and retraction.

Test No. V-004A was conducted using the increased hydraulic actuation pressure. The nozzle deployment system performed satisfactorily. The test sequence of events was as follows:

- Nozzle film coolant on $(\dot{w} = 0.5 \text{ lb/sec})$
- Engine start with the nozzle fully extended
- At engine start + 5 sec, the nozzle deployment system was signalled to retract the nozzle extension
- The nozzle extension was retracted in ≈ 2.25 sec
- At engine start + 11 sec, the nozzle deployment system was signalled to extend the nozzle extension

III, D, Test Data Analysis (cont.)

- The nozzle was extended in ≈1.1 sec
- At engine start + 15 sec, the nozzle film coolant was turned off
- At engine start + 25 sec, the nozzle deployment system was signalled to retract the nozzle extension
- The nozzle was retracted in ≈2.25 sec
- At engine start + 31 sec, the nozzle deployment system was signalled to extend the nozzle extension
- The nozzle was extended in 1.1 sec
- The test was terminated after 37 sec of steady-state chamber pressure operation

Not only had this test (Test No. V-004A) served to demonstrate operation of the nozzle deployment and retraction system, but it also demonstrated the capability of the static face seal to reseal the movable-to-fixed nozzle interface following the cyclic operation of the nozzle extension.

The nozzle performance data (I_{SP}) is presented in the performance summary, Table I, for Tests V-004 and V-004A. Specific impulse was calculated for both the film-cooled and uncooled portions of the tests and is presented for comparison in the table. The nozzle wall pressure data gathered during the nozzle retract and extend film-cooled cycle was used to calculate nozzle load. This calculated load was plotted vs. nozzle position. The plots are presented as Figures 18 and 19. These plots were made in an attempt to gain insight into the observed deviation in nozzle wall pressures from predicted as recorded on Tests V-001 and V-002 and its effect on nozzle load vs. position. It should be noted that the investigation of this pressure deviation was conducted in support of Test V-004 and V-004A and carried no further than as reported in this document.

The nozzle wall pressure deviations as measured (Figures 20, 21 and 22), were observed to increase to approximately 7 times that predicted (predicted ≈ 0.84 psia).

III, D, Test Data Anlysis (cont.)

Tests V-001, V-002, and V-003 were configured primarily to evaluate the piston seal and a secondary nozzle wall configuration different than that evaluated during Test Series IV. A comparison of the two 0.75 in. configurations is presented in Figure 23. To evaluate Configuration No. 2 0.75 in. wall discontinuity, thenozzle extension was fixed in a partially retracted position (retracted 2") and the engine fired. The plotted performance data, Figure 24, indicates an increased loss in nozzle performance over that experienced with Configuration No. 1 0.75 in. step height.

IV. SUGGESTED ADDITIONAL EFFORT

It is suggested that additional technical effort, supported by a subscale laboratory test program, be initiated to provide nozzle performance and thermal design data to supplement that obtained from the extendable/retractable nozzle test program and bridge the design gap as identified. This effort would coincide with and support significantly the NASA Space-Oriented Shuttle Program.

The objective of the extendable nozzles for space engines application program, was to provide design data which are applicable to space engines for which it is desired to upgrade performance by incorporating a high expansion ratio nozzle into a system restricted by vehicle or packaging requirements.

The data analysis phase of the experimental nozzle test program provided meaningful extendable/retractable nozzle design information, but it also identified additional areas of special concern to the performance analyst and engine designer. The main area of concern being the observed performance $(I_{sp(del)})$ loss associated with a nozzle wall discontinuity.

The performance analysis phase of this program (final report presentation) established that there is a definite advantage to incorporating an extendable nozzle into an engine system. This analysis considered performance losses resulting from wall discontinuities equal to zero. It is conceivable that with respect to the performance and payload gained by an extendable nozzle, which could be quite small when the performance loss associated with a given wall discontinuity is considered, that it is not economically feasible or worth the small gain in performance to encorporate an extendable nozzle into the engine system. This is particularly true when the effects of changing nozzle attachment area ratio and the configuration of the nozzle wall discontinuity on performance has not been firmly established or understood.

An attempt was made during analysis of the test data to correlate the effect of discontinuity and discontinuity location in the nozzle on nozzle performance loss.

IV, Suggested Additional Effort (cont.)

The conclusions drawn, mainly due to the limited experimental data available were (1) that the nozzle performance loss as measured could only be associated with the configuration tested, and (2) that yes, there is a loss in nozzle performance associated with a nozzle wall discontinuity, and that this loss increases with increasing step height of the discontinuity and possibly with a change in configuration of the nozzle wall discontinuity.

The proposed experimental laboratory program would not only provide an understanding of the performance loss problem, but would provide analytical data to satisfactorily evaluate the areas of design concern identified during evaluation of the test data. The three major areas of design concern are as follows: (1) how is nozzle performance effected by change in discontinuity, both axial and configuration; (2) how are nozzle heat loads effected by change in discontinuity, both axial and configuration; and (3) how effective is supersonic film cooling of the nozzle extension under these new conditions.

The experimental subscale laboratory test program would not only be configured to provide analytical data to effectively evaluate and satisfactorily answer the above areas of concern, but would also investigate the effects of nozzle wall pressure deviations as identified during Tests V-001 and V-002.

TABLE I. TEST DATA SUPPARY

Serios Objectives	Mossis performence data with continuous wall	uncooked Mozzla thermal and pressure data Evaluation of static face seal	Mozzle performance data both cooled and	uncooled with U.22-in. step neight Kossle theres! and pressure data	Evaluation of static face seal		Nozzle performance data both cooled and	uncooled with 0.35 in. meep height. Nogzle thermal and pressure data	Evaluation of static face seal		Nozzle performance data both cooled and	uncooled with configuration was a 0.75-in, step height	Nozzle thermal and pressure data	Evaluation of static face seat	Nozzle performance data uncooled and film	cooled Evaluation of Configuration No. 2 step	height = 0.75 fm.	Nozzle partially retracted & in.	benonstration of nozzle deployment and	retraction with engine firing Evaluation of the diametrial static seal	on the partially retracted tests gvaluation of the static face seal during demonatration of noszle deployment and	retraction
Norsle Position	Fully extended	Fully extended	Pully excended	Pully extended	Fully extended	Fully extended	Fully extended	Pully extended	Fully extended	Pully extended	Pully extended	Fully extended	Fully extended	Fully extended	Retracted 2 in.	Retracted 2 in.	Retracted 2 in.	Pully extended	Pully retracted	Fully extended	Fully extended Fully extended	Fully retracted Pully extended
Effective Step Height in.	0	•	0.25	0.25	0.25	0.25	0.50	0.30	0.50	0.50	0.75	0.75	0.75	0.75	0.75	0.75	0.75	0.50	0. 20	0.50	0.50	0.50
Step Height in.	•	•	0.25	0.25	0.25	0.25	0.50	0.50	0.50	0.50	0.75	0.75	0.75	0.75	0.30	0.50	0.50	0, 50	0, 50	0.50	0.50 0.50	0.50
,	298.8	297.8	297.0	294.7	291.8	285.9	296.3	294.4	289.9	286.2	295.6	293.9	289.8	285.6	233.3	290.9	292.3	286.8	271.0	286.2	284.6 292.8	278.9 292.4
**************************************	10.70	10.937	10.798	10.949	11.114	11.412	10.77	10.831	11.051	11.477	10.839	10.915	111.111	11.325	11.375	10.979	10.947	11.43	11.42	11.412	11.399	10.890
Per C	2	8	27	15	2	27	8	8	ଛ	8	8	9	3	8	Š	8	ಜ	77	13	8 8	2 2	* *
Deta Time	9-10	18-19	13-14	11-12	10-11	6-7	9-10	9-9	-	14-15	14-15	9-10	6-6	9-10	14-15	12-13	17-18	1	6-7	4-5	14-15	30-31 36-37
4 ta	•		19.160	19.165	19.172	19.175	19.120	_	19.135	_	-	19.097		19.113	19.085	19.074			19,06	19.055	19.055	19.055
- 4	3200	3257	3207	3226	3242	3264	3193	3195	3204	3285			3220	3235	3223	3194		3281	3099	3266 3081	3244	303/
*ec 3b/202	•	•	0	B 0.107	3 0.2526	5 0.5259			5 0.3176			4 0.0999									0.5045	
7 0 51			100.	100.		100.5		100.5		102.4		100.4		100.6				102.	101	101.01	100.6	3 8
뜊	~	1.711	1.664	1.685	1.667	1.660	1.635	1.618	1.614	1.728	1.694	1.666	1.660	1.673	1.706	1.689	1.731	1.681	1.693	1.697	1.745	1.806
* t	10.70	10.937	10.798	10.942	10.961	10.886	10.111	10.752	10.733	10.988	10.839	10.815	10.816	10.826	10.873	10.875	10.947	10.94	10.93	10.909	10.895	10.875
\$ 0 1 <u>b/eec</u>	6.713	6.902	6.775	6.805	6.789	6.78		6.645	6.627	6.961	6.817	6.759	6.750	6.111	6.855	6.831	6.938	6.86	6.87	6.906 6.909	6.926	6.999
\$ 15/80C	3.995	4.035	4.023	4.037	4.072	4.092	4.090	4.107	4.106	4.027	4.022	4.056	4.066	4.049	4.018	4.044	4.009	8.4	4 .08	4.045	7.5	3.876
Test	3-3-71	3-4-11	2-19-71	2-19-71	2-15-21	2-19-71	3-4-21	17-9-1	3-1-71	11-6-6	3-25-71	3-25-71	3-25-71	3-25-71	4-2-11	4-2-71	4-2-11	++11		17-11-4		
7. 6. Yo	00	*002A	100	A002A	600	\$00°C	100	200	8	ğ	100	•003	600	7 00	100	8	ĝ	켷		00¢		
Took			==				171				N.				>							

TABLE II

EXTENDABLE NOZZLE ENGINE TEST SUMMARY

ed by Conditions and Remarks	TD-185-001 $1/4$ " step, $1/2$ 1b/sec GN_2 flowrate. Engine & TCR #1 valves did not open dur to procedure error. Repeated as $001A$.	TD-185-001 $1/4$ " step, high GN_2 flowrate. Nominal LM ullage & TCR #1 pressures (183 psia) gave above-nominal EXNOZ engine pressures and flowrates.	target tank pressures to give nominal engine operation. Fuel pressure OK at 169 psia, oxidizer tank pressure dropped 13 psi during run. GN ₂ manifold pressure rose 50% during firing.	TD-185-001 Repeat of 2-004 above, with same results. After & TCR's #2 test, replaced oxidizer regulators, installed gas and 3 flowmeter and P-T sensors in ${\rm GN}_2$ system, made ${\rm GN}_2$ flowrate vs. manifold pressure checkout.	TD-185-001 Repeat of 2-004: $1/4$ " step, high GN ₂ flow. Pro- & TCR #3 cedure error stopped tank pressurization at +5 seconds,
:ion Defined				sec. TD-185-(& TCR's and 3	
No. Duration	. 0 sec.	A 3 sec.	15 sec.	15	B 15 sec.
Test No.	11 1-001	71 1-001A	-71 2-004	-71 2-004A	-71 2-004B
Date	2-9-71	2-9-71	2-11-71	2-11-71	2-19-71

ran in blowdown until shutdown.

Table II Sheet 1 of 4

Repeat of 2-004B above. Oxidizer tank pressure still dropped. GN_2 manifold pressure rose 50% as before, 1/4" step, uncooled. After test performed in-place Pressure 1/4" step, low 0.1 lb/sec GN_2 flowrate. Coolant cycle oxidizer pressurization valve to prevent 1/2" step, high (0.5 lb/sec) GN_2 flow. Had to calibration on thermocouples and calorimeters. Repeat of . '02 above for data repeatibility. 1/4" step, intermediate 0.3 lb/sec GN_2 flow. but ${
m GN}_2$ flowmeter showed constant rate. Installed 1/2" step ring after test. Zero step, uncooled baseline test. Installed zero step height ring flowrate too low (0.067 lb/sec) rise apparently thermal effect. tank overpressurization. Conditions and Remarks Repeat of 2-002 above. Defined by TD-185-001 TD-185-001 TD-185-001 TD-185-001 TD-185-001 TD-185-001 TD-185-001 TD-185-001 & TCR #3 & TCR #4 Extendable Nozzle Engine Test Summary (cont.) Duration 15 sec 15 sec 20 sec 15 sec 15 sec 15 sec 30 sec 20 sec Test No 2-004C 2-002A 1-002A 2-005 1-002 2-001 3-004 2-003 2-19-71 2-19-71 2-19-71 2-19-71 2-19-71 3-3-71 3-9-71 3-4-71 Date

Table II
Sheet 2 of 4

Extendable NozzleEngine Test Summary (cont.)

Date	Test No.	Duration	Defined by	Conditions and Remarks
3-9-71	3-003	30 sec	TD-185-001	$1/2"$ step, 0.3 lb/sec ${ m GN}_2$ flowrate.
3-9-71	3-002	30 sec.	TD-185-001	$1/2"$ step, 0.1 lb/sec ${ m GN}_2$ flowrate.
3-9-71	3-001	30 sec	TD-185-001	1/2" step, uncooled. Installed $3/4$ " step and boundary layer temperature probes after test.
3-25-71	4-004	30 sec	TD-185-001	3/4" step, 0.5 lb/sec GN ₂ flow.
3-25-71	4-003	30 sec	TD-185-001	$3/4$ " step, 0.3 lb/sec GN_2 flow.
3-25-71	4-002	30 sec	TD-185-001 & TCR #5	$3/4"$ step, 0.1 lb/sec ${ m GN}_2$ flowrate.
3-25-71	4-001	30 sec	TD-185-001 & TCR #6	$3/4"$ step, uncooled. Measured throat (19.09 in. 2) and replaced actuators.
4-2-71	5-001	30 sec	TD-185-001 & TCR #6	$1/2"$ step, extendable skirt retracted 2 inches throughout firing, high ${ m GN}_2$ coolant flow. Cycled oxidizer pressure valve to prevent overpressurization.
4-2-71	5-002	30 sec	TD-185-001 & TCR #6	$1/2"$ step, skirt retracted 2 inches, low 0.1 lb/sec ${ m GN}_2$ flow.
4-2-71	5-003	30 sec	TD-185-001 & TCR #6	1/2" step, skirt retracted 2 inches, uncooled.

Table II Sheet 3 of 4

Extendable Nozzle Engine Test Summary (cont.)

Defined by Conditions and Remarks	01	unpowered between actualisms. 185-001 1/2" step, retracted and extended nozzle twice
<u>Duration</u> <u>Defin</u>	17 sec TD-185-0	38 sec TD-18
rest No.	2 -00%	5-004A
Date	4-6-71	4-14-71

TOTAL DURATION 548

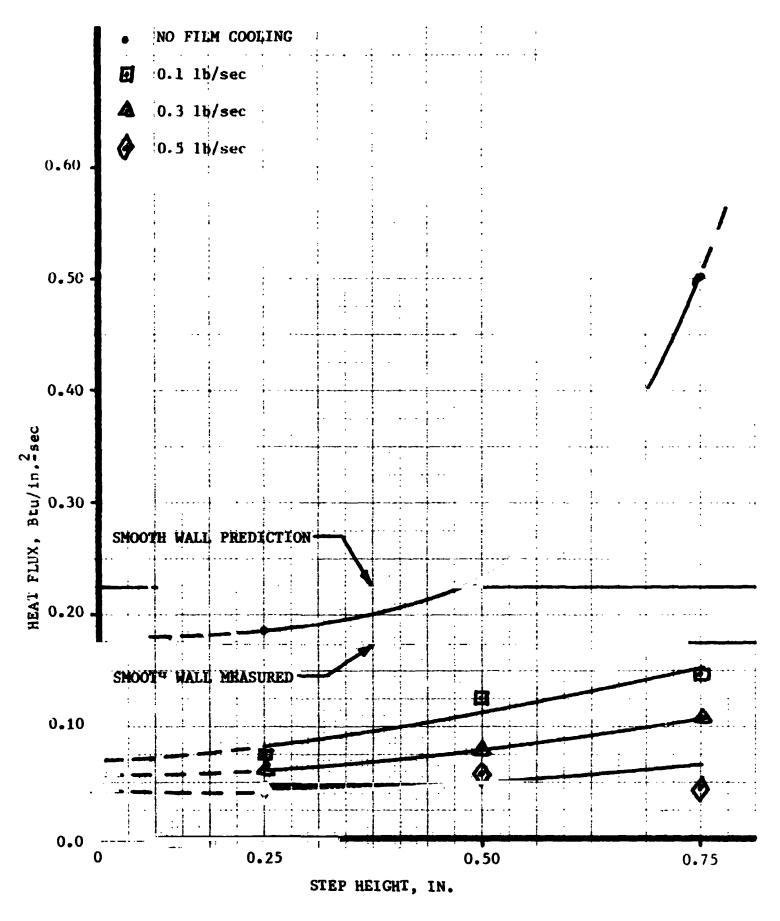
during firing, once with high ${\rm GN}_2$ ccolent flow,

once uncooled.

& TCR #7

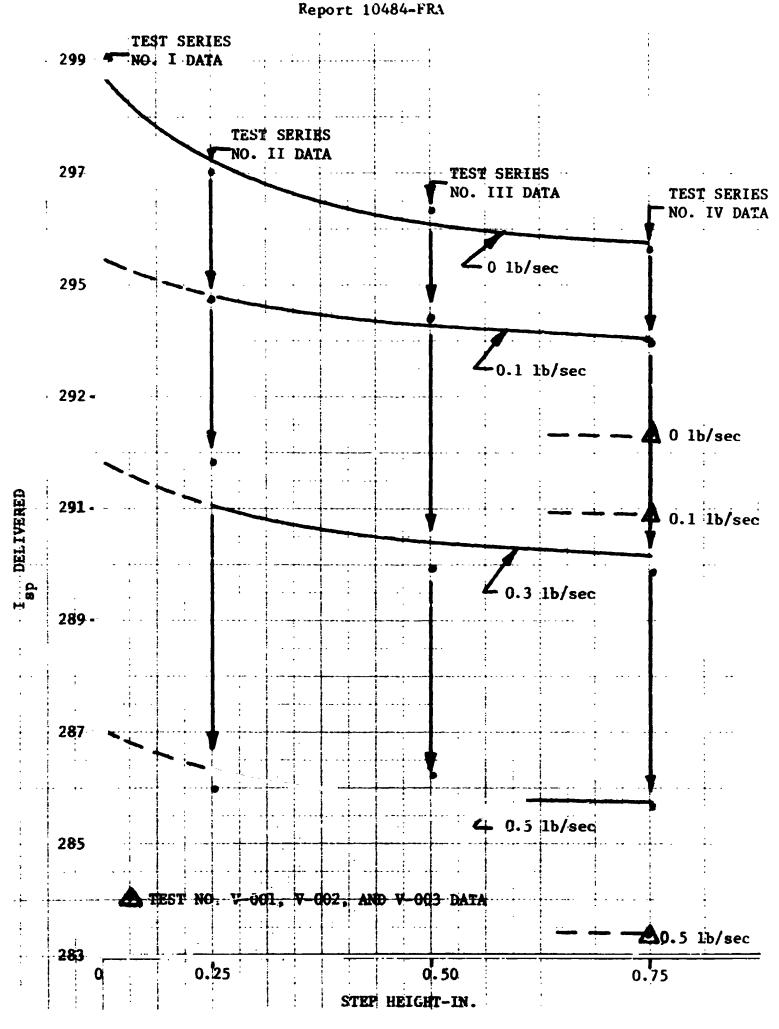
TABLE III
THERMAL INSTRUMENTATION SUMMARY

Axial Distance,	Chamber	Area	Thermo	couple	Calorin	neter	Gas P	robe
Inch	Dia. in.	Ratio	Number	Angle	Number	Angle	Number	Angle.
1.567	22.375	Step Region	1	6	1	0		
			2	126				
			3	246				
2.097	22.375	Step Region	4	18	2	120	2	120
			5	138	3	240		
			6	258				
2.447	22.375	Step Region	7	6	4	132	3	
			8	126	5	252		
			9	246				
2.787	22.375	Step Region	10	18	6	0		
			11	138				
			12	258				
3.417	22.68	21.1	13	6	7	12	4	120
			14	126	8	120		
			15	246	Q	240		
4.167	23.24	22.3	16	18				
			17	138				
			18	258				
9.267	26.52	28.9	19	6				
			20	126				
			21	246				
13.167	28.76	33.9	22	6				
			23	126				
			24	246				

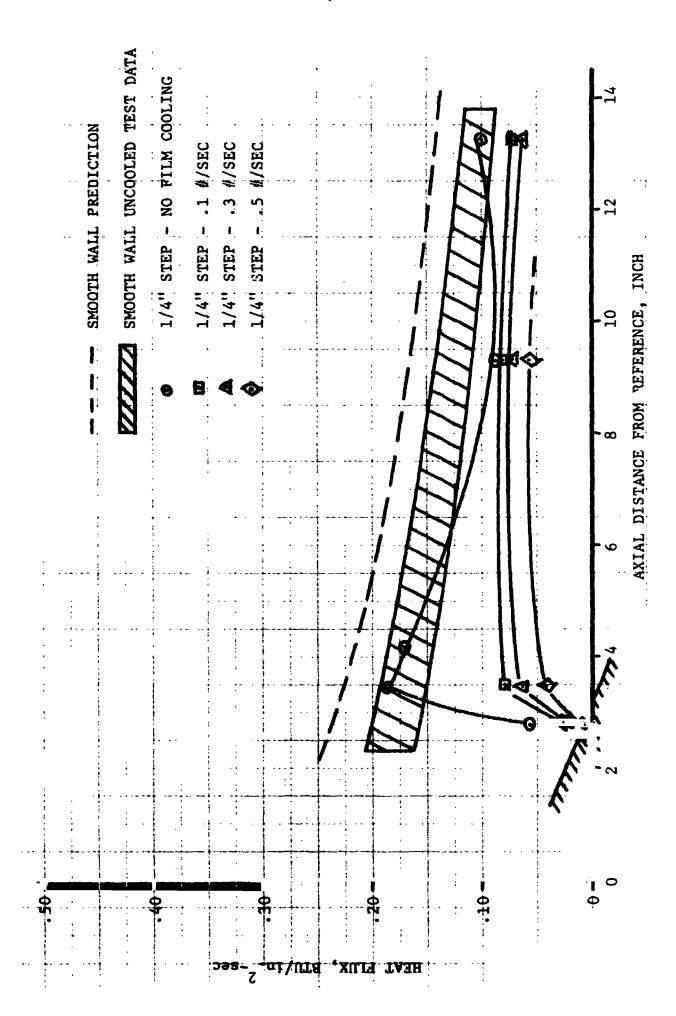


Heat Flux vs Step Height

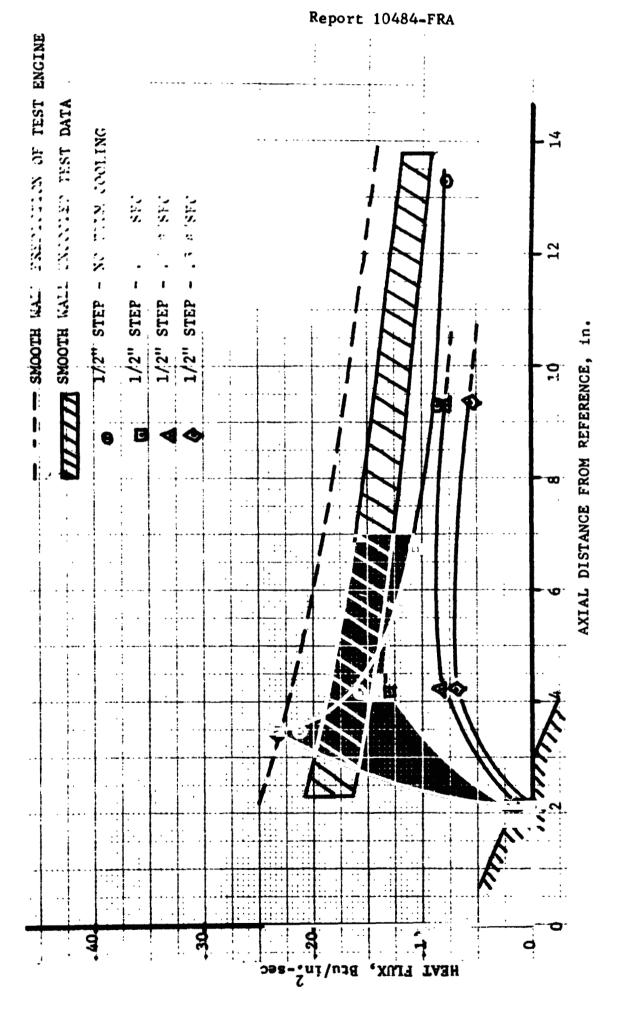
Figure 1



Delivered I vs Step Height
Figure 2



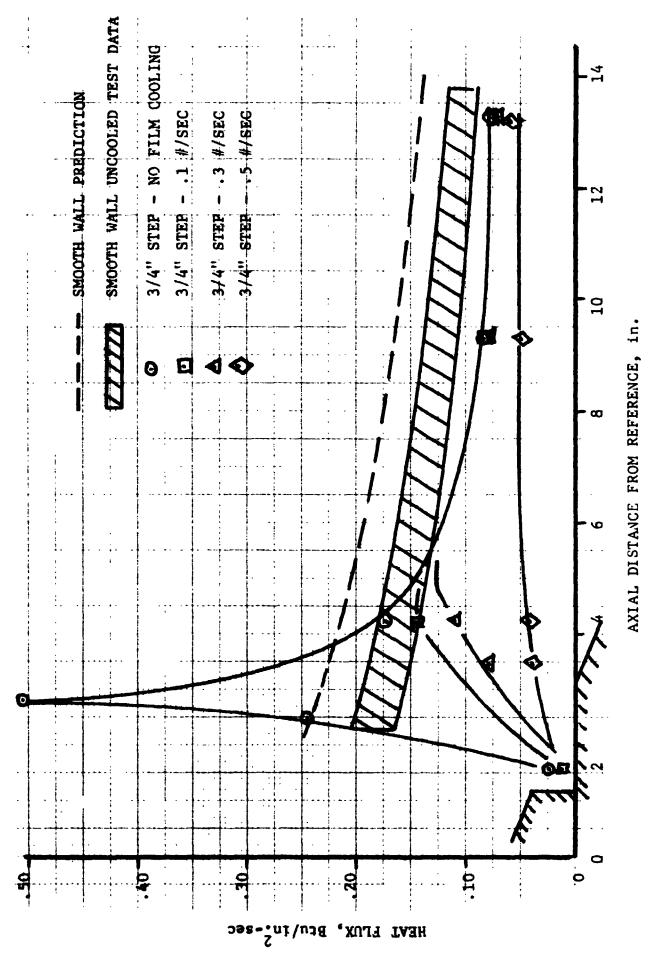
Test Series II Heat Flux Data: 0.25-in. Step



Test Series III Heat Flux Data: 0.50-in. Step

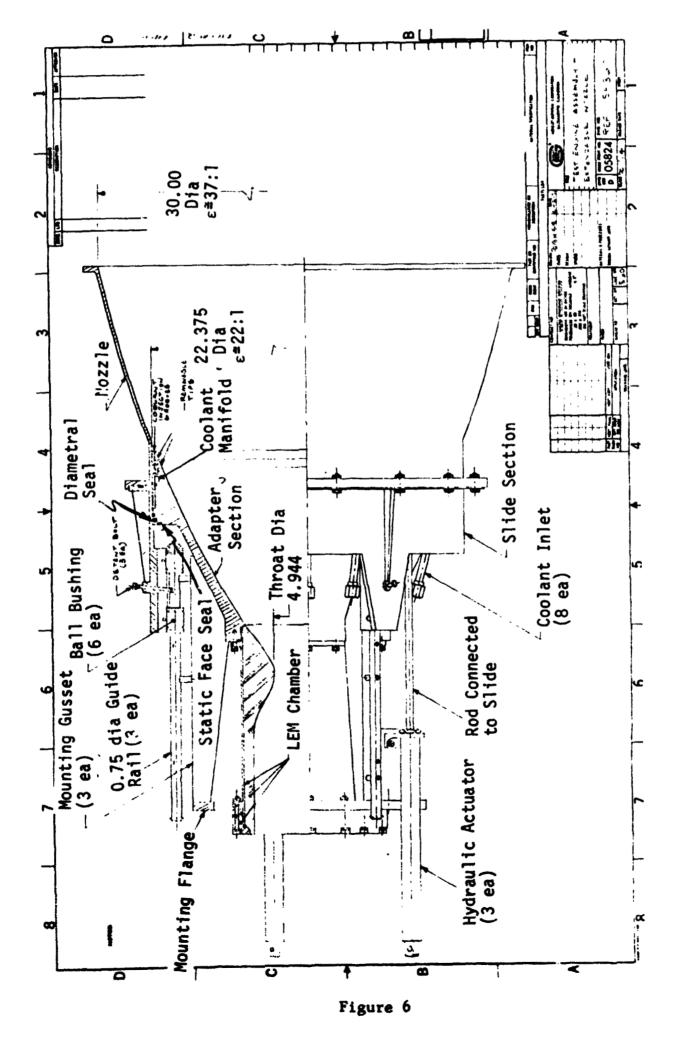
Figure 4

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Test Series IV Heat Flux Data: 0.75-in. Step

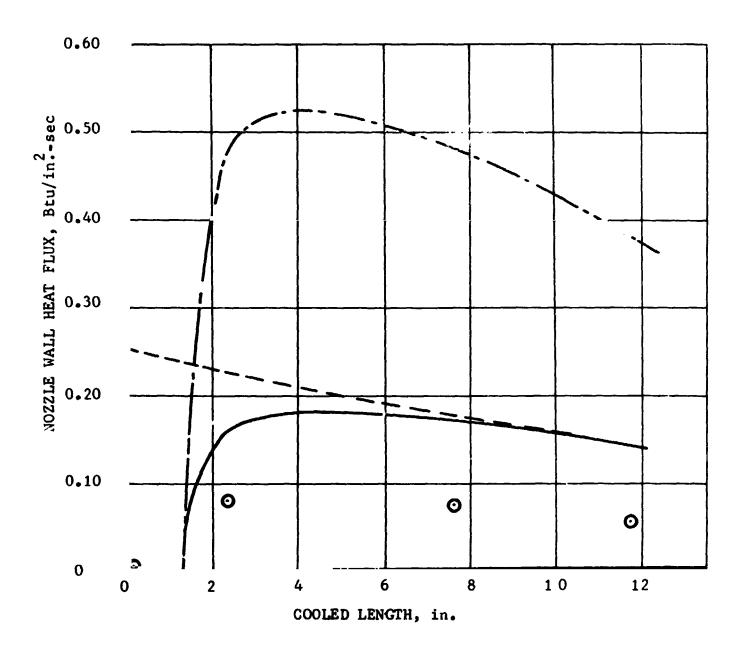
Figure 5



Extendable Nozzle Test Engine Assembly

Report 10484-FRA

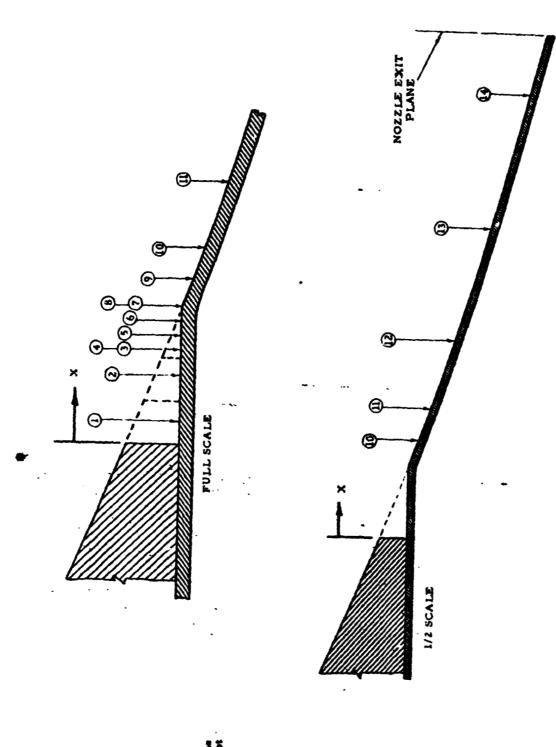
Design Guide Prediction with 3.0 Amplification Factor
Design Guide Prediction with 1.0 Amplification Factor
Smooth Wall Prediction
Series III, Test -003 Data (0.5-in. Step, 0.318 lb/sec
GN₂ Film Cooling)



Nozzle Wall Heat Flux with Film Cooling

Figure 7

Report 10484-FRA



Nozzle Wall Pressure Taps for Test Engine

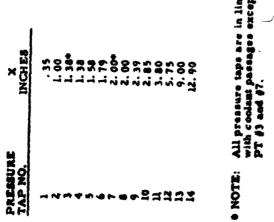
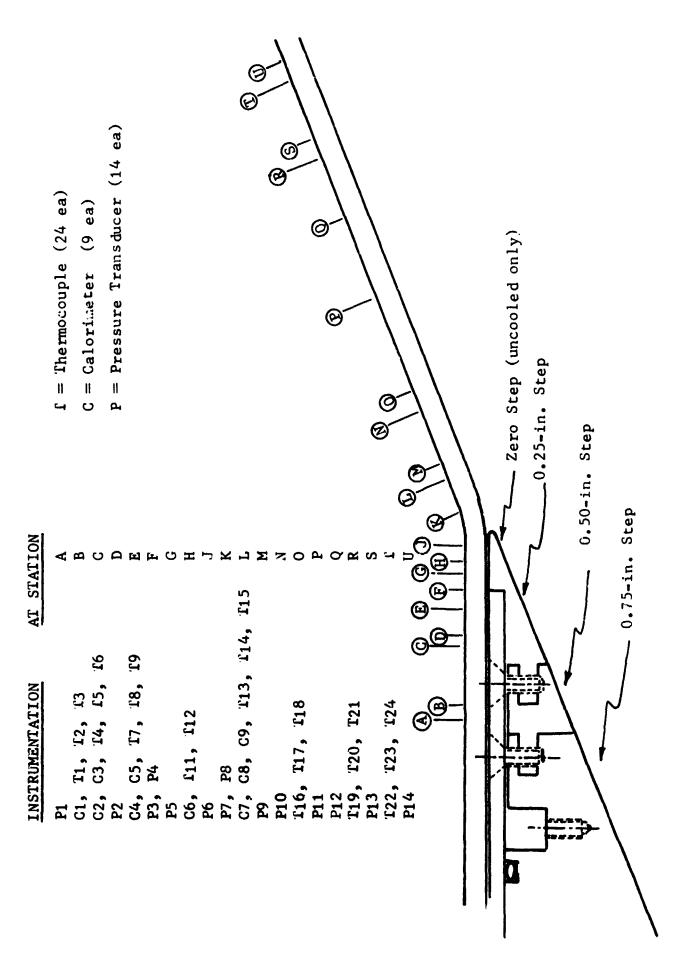
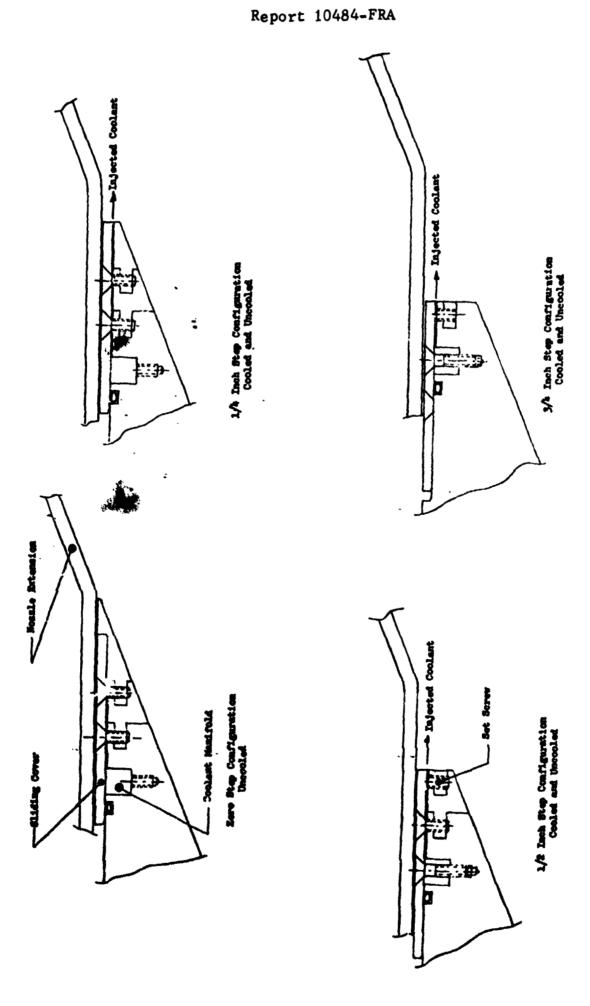


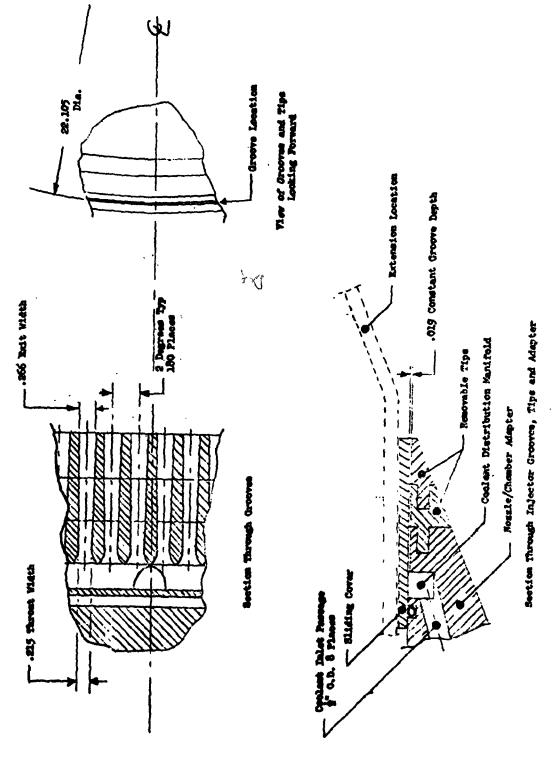
Figure 8



Instrumentation in Region of Removable Tips

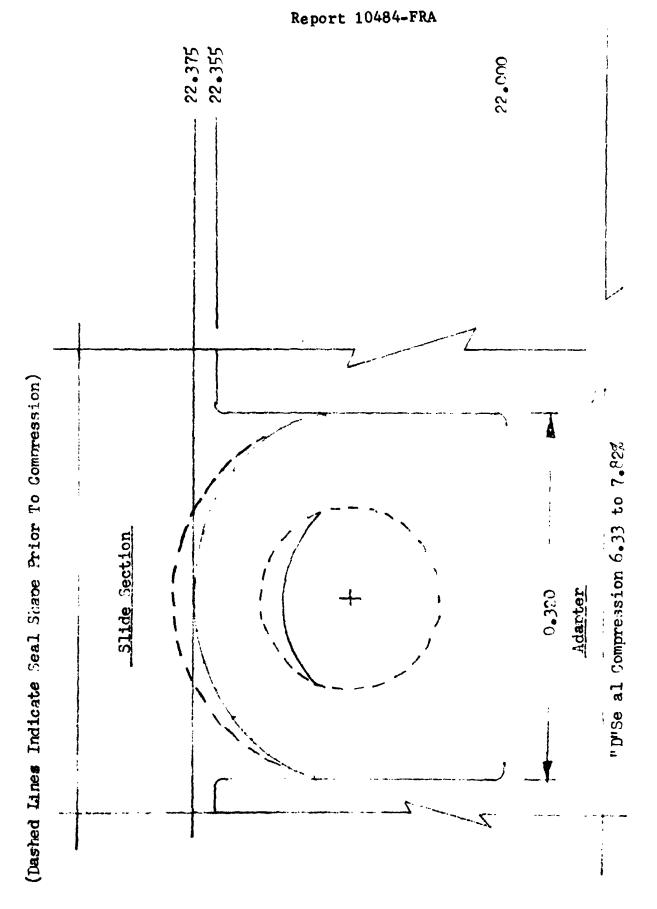


Variable Step Removable Tip Concept

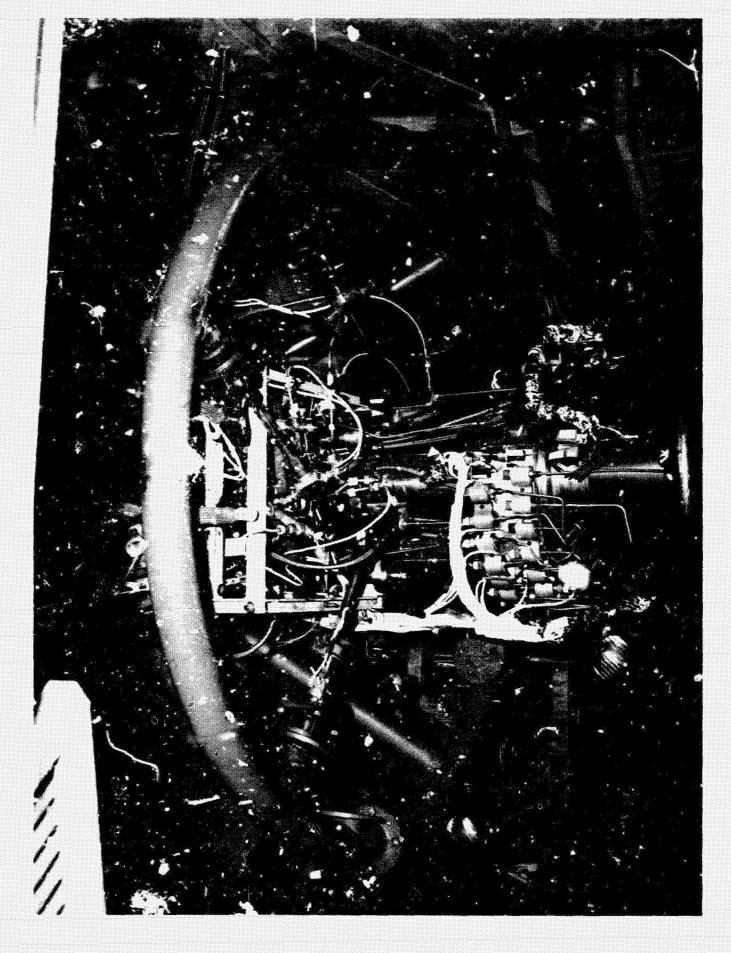


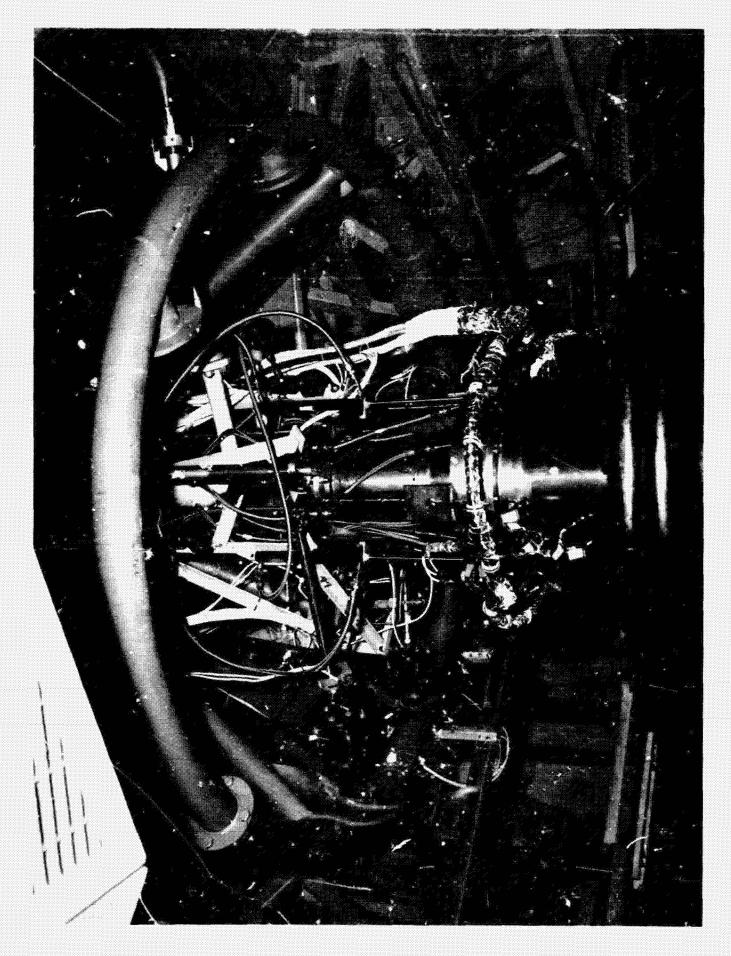
Supersonic Coolant Injector Grooves

Figure 11



"D" Ring Tubular Seal as Used in the Extendable Nozzle Test Engine

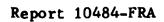


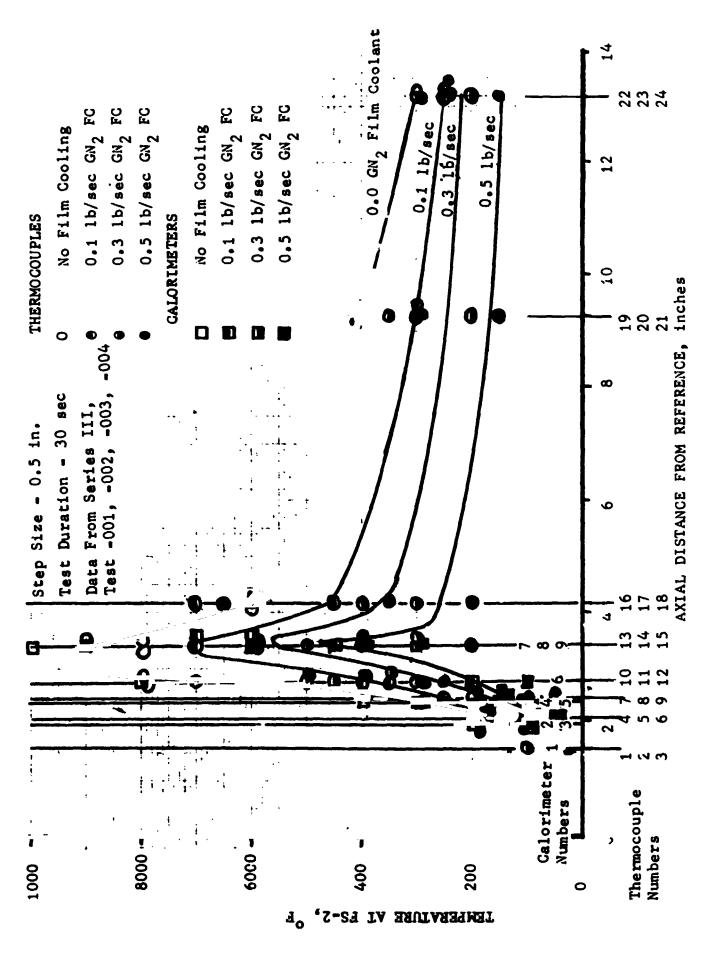


Engine Installed in Stand, Showing Thermal Instrumentation

Figure 14

Engine Installed in Stand, Showing LEM Ascent Portion





Extendable Nozzle Temperature Summary

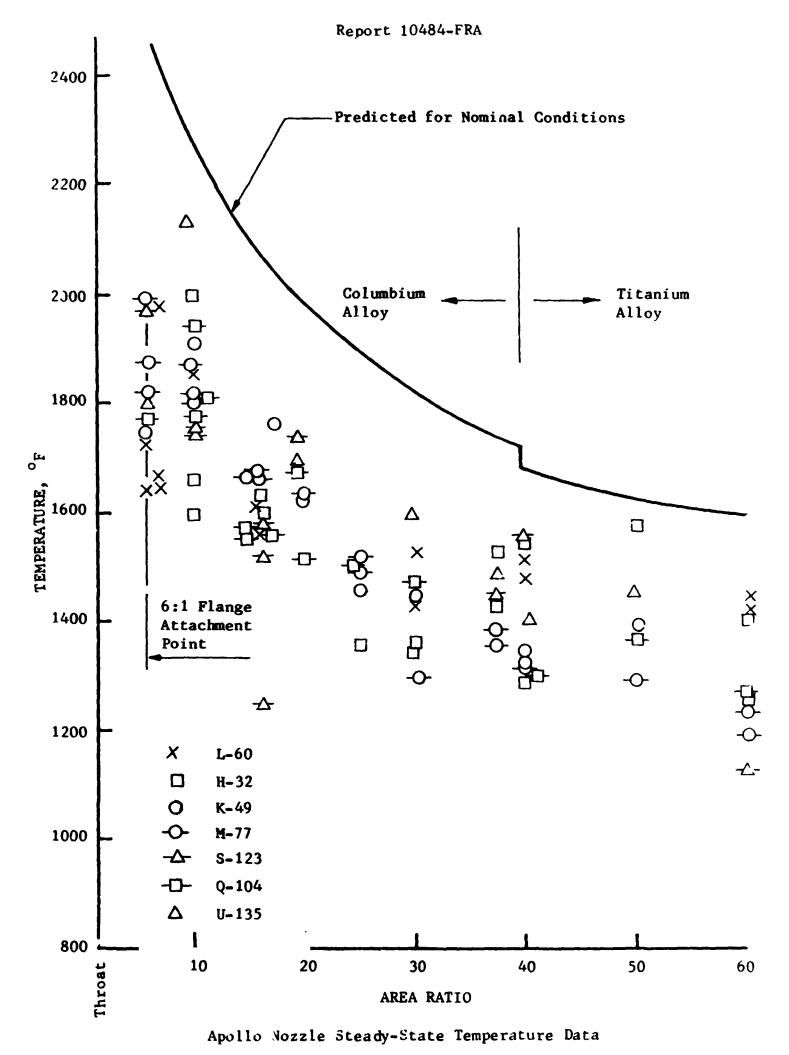
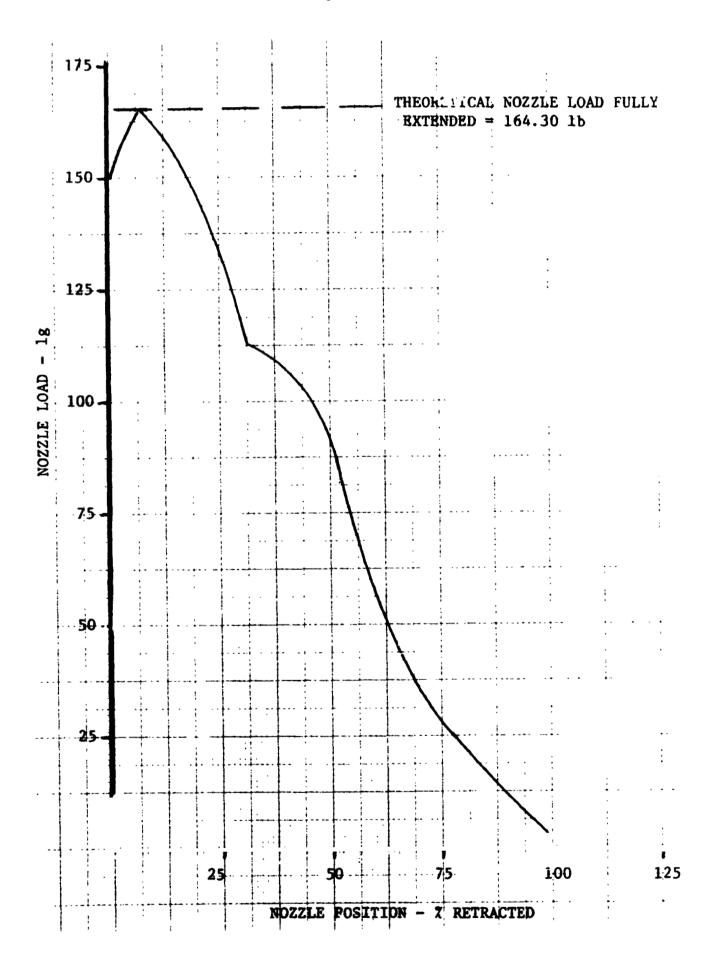


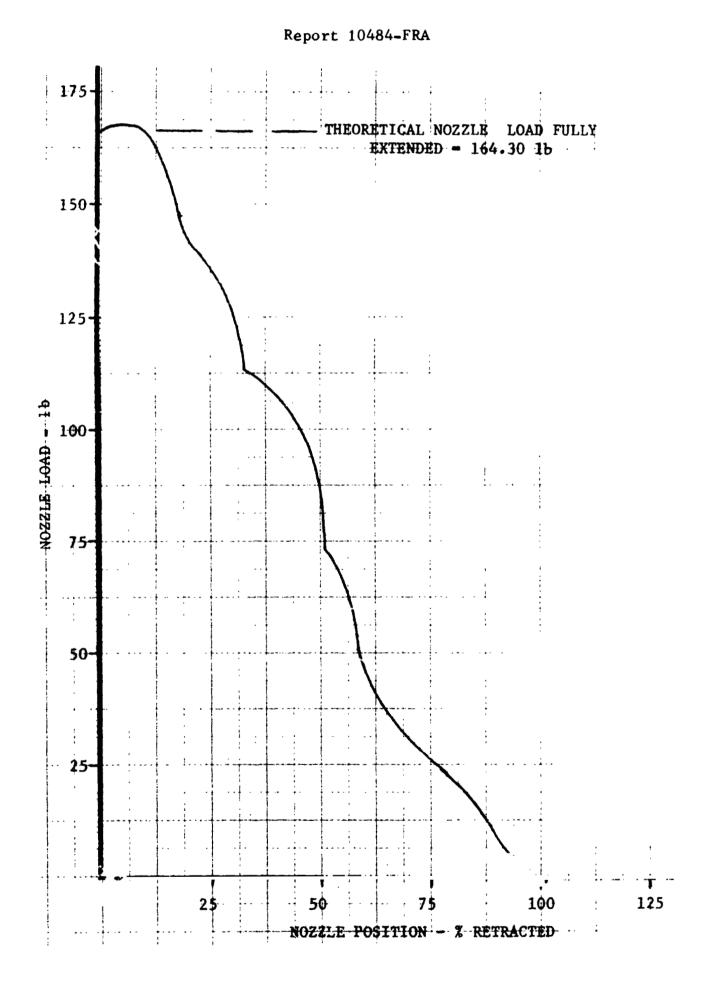
Figure 17

Report 10484-FRA



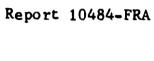
~ e Load vs Nozzle Position - Extend Cycle

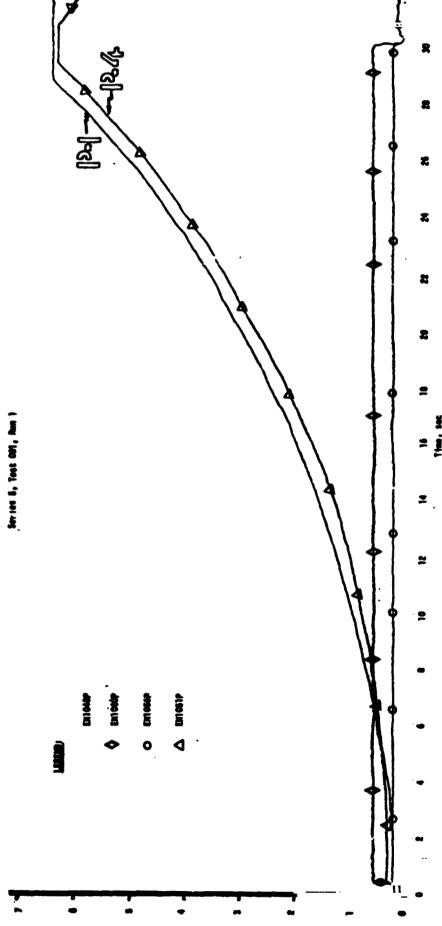
Figure 18



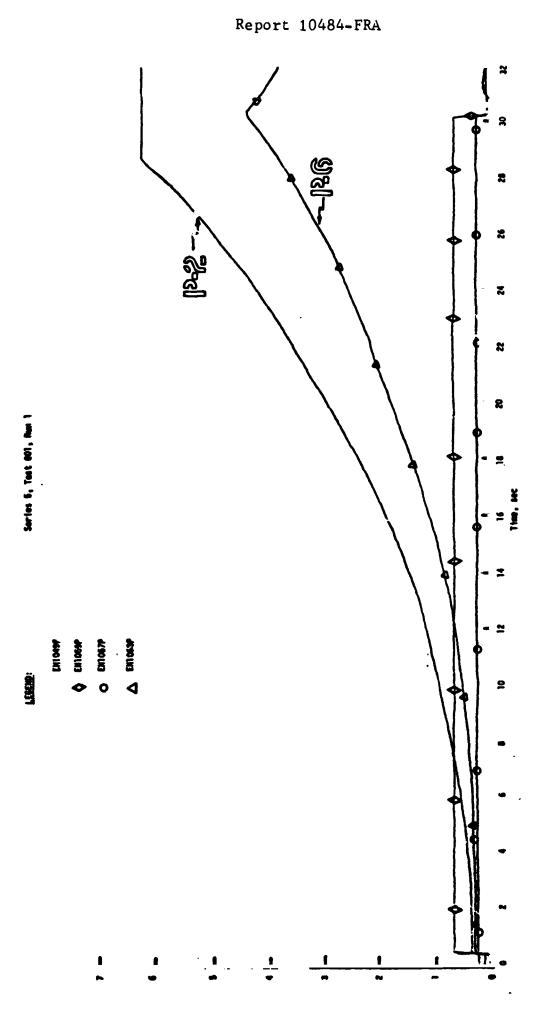
Nozzle Load vs Nozzle Position - Retract Cycle

Figure 19



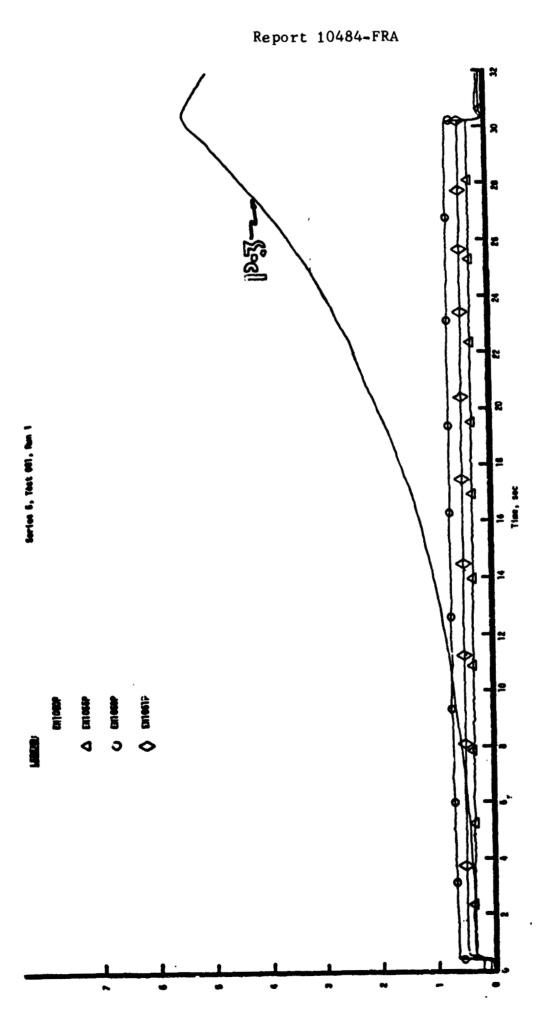


Nozzle Wall Pressures (Transducers P-1, P-4) vs Time



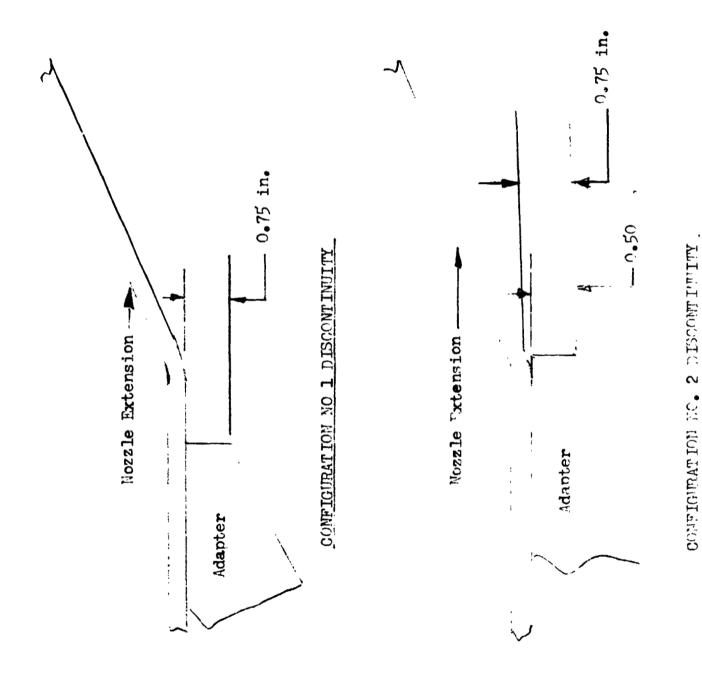
Nozzle Wall Pressures (Transducers P-2, P-6) vs Time

Figure 21

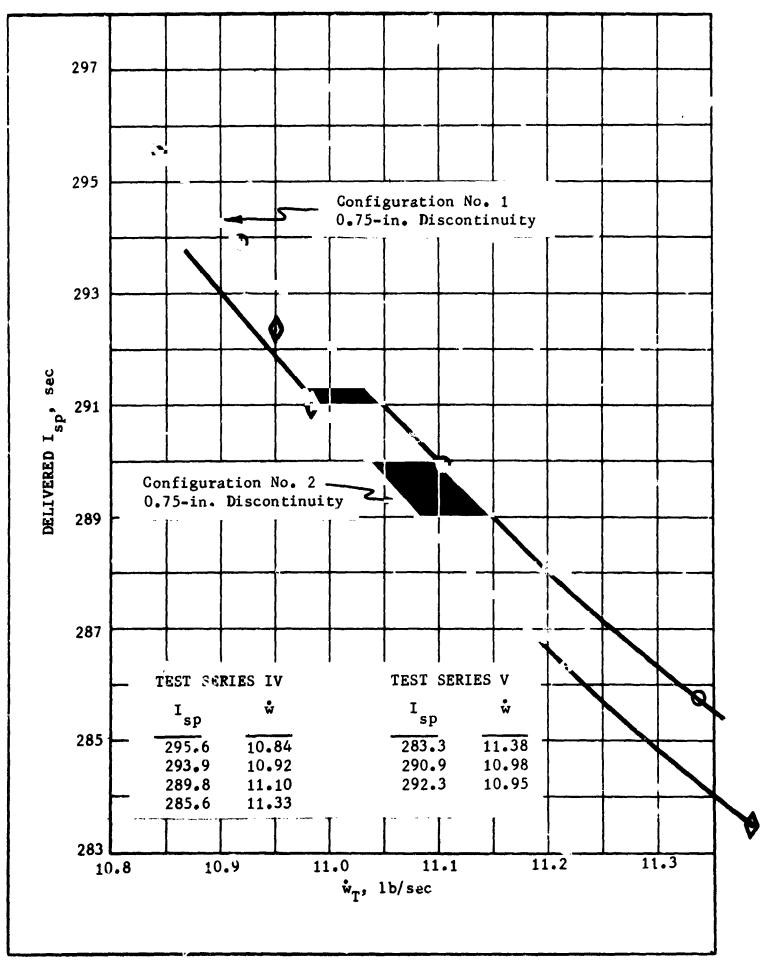


Nozzle Wall Pressures (Transducer P-3) vs Time

Figure 22



Comparison of 0.75-in. Nozzle Wall Discontinuities



Effect of Configuration on Performance (0.75-in. Step)

Figure 24